

Sound Intensity & Human Hearing:

The human ear is exquisitely sensitive to detecting sounds (*n.b.* dogs hear $\sim 100\times$ better!) Due to this large dynamic range, our ears have an \sim logarithmic response to sound intensity.

Sound Intensity, I (*RMS Watts/m²*) $\propto \xi_{rms}^2$ (*RMS particle displacement amplitude*)²
 as well as: $I \propto u_{rms}^2$ (*RMS particle velocity amplitude*)²
 as well as: $I \propto p_{rms}^2$ (*RMS over-pressure amplitude*)²

The relationship of the {magnitude of the} *RMS* sound intensity at a given point \vec{r} in a sound field associated *e.g.* with monochromatic/pure-tone sine-type travelling plane waves propagating in free air (*i.e.* the great outdoors) to the above quantities is:

$$I_{rms}(\vec{r}, t) = p_{rms}^2(\vec{r}, t) / \rho_o c = \rho_o c \cdot u_{rms}^2(\vec{r}, t) = \rho_o c \cdot \omega^2 \xi_{rms}^2(\vec{r}, t) = c w_{rms}^{ac}(\vec{r}, t)$$

where:

$p_{rms}(\vec{r}, t)$ = *RMS over-pressure amplitude (RMS Pascals)* at the point \vec{r}
 ρ_o = equilibrium mass density of dry air (@ *NTP*) = 1.204 kg/m³
 $c = v_{air}$ = longitudinal propagation speed of sound in dry air = 343 m/s (@ *NTP*)
 $u_{rms}(\vec{r}, t)$ = so-called *RMS longitudinal particle velocity amplitude (RMS m/s)* at the point \vec{r}
 $\omega = 2\pi f$ = angular frequency (*radians/sec*), f = frequency (*Hz*)
 $\xi_{rms}(\vec{r}, t)$ = *RMS longitudinal particle displacement amplitude (RMS meters)* at the point \vec{r} .

The *RMS* acoustic energy density (*Joules/m³*) at the point \vec{r} is given by:

$$w_{rms}^{ac}(\vec{r}, t) = p_{rms}^2(\vec{r}, t) / \rho_o c^2 = \rho_o u_{rms}^2(\vec{r}, t) = \rho_o \omega^2 \xi_{rms}^2(\vec{r}, t)$$

The quantity $z(\vec{r}) \equiv p_{rms}(\vec{r}) / u_{rms}(\vec{r})$ (*Pa-s/m*) is known as the specific acoustic impedance of the medium, which for monochromatic plane waves propagating in a free air sound field is a constant (*i.e.* independent of frequency): $z_{ac}^{free\ air} \equiv z_o = \rho_o c \approx 415 \text{ kg/m}^2\text{-sec} = 415 \Omega_{ac}$ (@ *NTP*).

Thus, we also see that for such plane waves propagating in a free-air sound field that:

$$I_{rms}(\vec{r}, t) = p_{rms}(\vec{r}, t) \cdot u_{rms}(\vec{r}, t) = p_{rms}^2(\vec{r}, t) / z(\vec{r}) = u_{rms}^2(\vec{r}, t) \cdot z(\vec{r}) = c w_{rms}^{ac}(\vec{r}, t).$$

Note that *RMS* quantities are frequently/often used in acoustical (and other) physics because for a pure-tone/single-frequency sine wave-type sound, they conveniently correspond to the time-averaged value of that quantity:

$$\langle I \rangle = \frac{1}{\tau} \int_{\text{one cycle}} I(t) dt = \frac{1}{2} I_{peak} = I_{rms} \propto \frac{1}{\tau} \int_{\text{one cycle}} p^2(t) dt = \frac{1}{\tau} p_o^2 \int_{\text{one cycle}} \sin^2 \omega t dt = \frac{1}{\tau} \frac{1}{2} p_o^2 = \frac{1}{\tau} p_{rms}^2$$

p_{rms} = *RMS (Root-Mean-Square) of sound (over-)pressure amplitude (RMS Pascals = RMS Newtons/m²)*

For a pure-tone (*i.e.* single-frequency) sine wave: $p_{rms} \equiv \frac{1}{\sqrt{2}} p_o \{ \Rightarrow p_{rms}^2 \equiv \frac{1}{2} p_o^2 = \langle p_o^2 \rangle \}$